# EFRC Workshop Pulsations Effects on valve performance

### Gunther Machu HOERBIGER Compression Technology



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## Working principle of compressor valves



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### **Overview different valve designs**







**Steel plates** 

Plastic profiled rings

**Plastic plates** 



Poppet valves



Steel reed valves



Flapper valves



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### **Overview different valve designs**



**Steel plates** 



Plastic plates / profiled rings



suction valve = intake





delivery valve = outlet





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### **Overview different valve designs**



**Steel reed valves** 





Poppet valves







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## Working principle of compressor valves



## Valve dynamics

Flow mechanics – valve / ventilation losses



### Valves – power losses

Power Losses

Pressure loss

$$\Delta p \approx \rho (\frac{V}{\alpha_{\Phi} \cdot \Sigma A_{Ventil}})^2$$

Similar valve clearance

$$(\Sigma A_{Ventil})^{\frac{3}{2}} \approx D^2 \cdot H \approx \dot{V} \cdot n^{-1}$$
  
Pressure loss for given  
delivery  $\dot{V}$ 

$$\Delta p \approx \rho \cdot n^{\frac{4}{3}} \cdot \alpha_{\Phi}^{-2}$$





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### Efficiency coefficient for different designs





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## **Sealing element dynamics**

The impact velocity of the sealing element against the guard / seat is the most important design criteria for a compressor valve!

Stresses in the







are directly proportional to the impact speeds!

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# Spring design

#### - Dynamic & static spring criteria

- Solid height stress
- Individual coil contact
- Lift off





Dynamic stresses directly proportional to impact speed

Pulsations can lead to increased impact speeds, increasing the loading of the springs!



# In – cylinder pulsations

#### Industrial standard:

➢ Theory of M. Costagliola (1949) – assuming homogeneous cylinder pressure everywhere in the cylinder

#### Advanced model

Pressure depends on location -1D model of wave propagation inside



### Validation with measurements

#### **Propane application:**

2nd stage, 346.1 [mm] bore, 139.7[mm] stroke, 1190[rpm], 3.9 – 12.6[bara]

*Ind. pressure at centerscrew of suction valve (measurement position 1)* 





Indicated pressure at centerscrew of discharge valve (measurement position 2)

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### Validation with measurements

**Propane application:** 

EFRC

2nd stage, 346.1 [mm] bore, 139.7[mm] stroke, 1190[rpm], 3.9 – 12.6[bara]



### **Effect on compressor performance**





Note: suction losses appear quite usual, discharge losses ~ 3 times higher

### Validation with measurements

#### **Natural Gas Application:**

1st stage, 387.4 [mm] bore, 76.2[mm] stroke, 1790[rpm], 2 – 5.3[bara]



### Line pulsations: effects on valve dynamics

#### **Question:**

How would a 7% peak to peak line pulsation affect valve dynamics?

(MW = 28.22 kg/kmol,  $p_s = 1.5bara, p_d = 6bara$ )

#### Original valve design – no pulsations:

- no late closures
- Impact within design limits

#### With pulsations:

Late closing of discharge valve, increased closing speed

Increased valve flutter – increased wear





### Now, what can be done

Measures to reduce impact speeds: advanced designs



ring valves with sync. plates



#### Advanced nonmetallic sealing elements:

- can withstand higher impact speeds







# Summary

• Although it seems simple, valve design and the effect of pulsations requires a lot of facts to be considered!

### Understanding of

- Flow mechanics
- Valve dynamics
- Spring dynamics
- Sealing element material characteristics

### mandatory!



### Thank you!

